

Comparative Analysis of Free and Forced Convection for Aluminum and Copper-Made Radiator Tube

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Abstract: When it comes to mechanical characteristics, banana fibre is second to none, and it's abundant in nature. Efficient cooling can extend the life of the engine and enhance its performance. In order to evaluate the relative heat dissipation capabilities of aluminium and copper radiator tubes, this project calls for the construction of a flow heat exchanger. We calculated heat transfer and total efficiency using the log mean temperature difference approach. The ANSYS software was also used for the thermal behaviour investigation. A cooling fan is important to this project's simulation of real-time conditions for moving and idle variables (forced and free convection). Both theoretical and experimental investigations into cross-flow heat exchangers have yielded the following results: the heat transfer and overall heat transfer coefficients are determined by using the log mean temperature difference method, which is dependent on the inlet and outlet fluid temperatures as well as the effectiveness of the device. This finding demonstrates that heat transfer is linearly proportional to the mass flow rate of air and that its effectiveness is inversely proportional to the mass flow rate.

Keywords: Ansys Fluent; Convection Heat Transfer; Heat Exchanger; Engine Oil SAE 5w30; Anemometer; Speed Control Regulator

Introduction

The internal combustion engines found in modern automobiles produce an enormous quantity of thermal energy. The combustion chamber is responsible for producing this heat by lighting the fuel and air combination [6]. The internal combustion engine generates thrust by the explosion of the piston, which in turn turns the crankshaft by pulling on the connecting rods [7-13]. In the vicinity of the combustion chamber, metals can reach temperatures higher than 1000°F. Efficient heat disposal is

essential to avoid engine oil, cylinder walls, pistons, valves, and other components from being overheated by these extremely high temperatures. The mechanical work consumes 25% of the heat, the exhaust gases lose 35%, and the lubrication system loses 5%. At least 35% of the heat should be dissipated by the cooling system [15-19].

In favour of the more efficient liquid-cooled system, automotive engines have abandoned the air-cooled one [20]. A heat-absorbing coolant runs throughout the engine in a liquid-cooled system, particularly in the cylinder head area of the engine block, to dissipate heat, particularly around the combustion chamber. After soaking up the heat from combustion, the coolant is pushed back out into the atmosphere via the radiator after going through the engine. After cooling, the liquid is transported to the engine for recirculation [21-25]. Heat exchangers can be found in many different types of machinery and appliances, including: boilers, condensers, evaporators, regenerators, oil coolers for heat engines, milk chillers in pasteurising facilities, radiators in automobiles, and many more industrial operations [26]. When the intake and outlet conditions are determined, the logarithmic mean temperature difference (LMTD) can be used to build heat exchangers. To simplify the analysis of finding the inlet or exit temperature of a specific heat exchanger, one can use a method that relies on the heat exchanger's effectiveness (a concept initially put forward by Nusselt) and the number of transfer units (NTU). The effectiveness of a heat exchanger is defined as the ratio of its actual heat transfer to its maximum possible heat transfer [27-33].

An apparatus that allows the transfer of thermal energy between fluids of various temperatures is known as a heat exchanger. The flow of coolant through the tube and the airflow over the fins. The two fluids are kept separate as they flow at right angles to one another. As a result, heat is transferred from the coolant to the air. In cross-flow heat exchangers, this flow is used [34-41]. The term "cross-flow heat exchanger" refers to a piece of machinery that allows heat to be transferred between fluids of different temperatures [42]. Typically, in a cross-flow heat exchanger, the two streams of fluid travel in opposite directions. Improving the number of turns in a cross-flow heat exchanger decreases the oil flow rate while increasing the heat transfer (Figure 1).

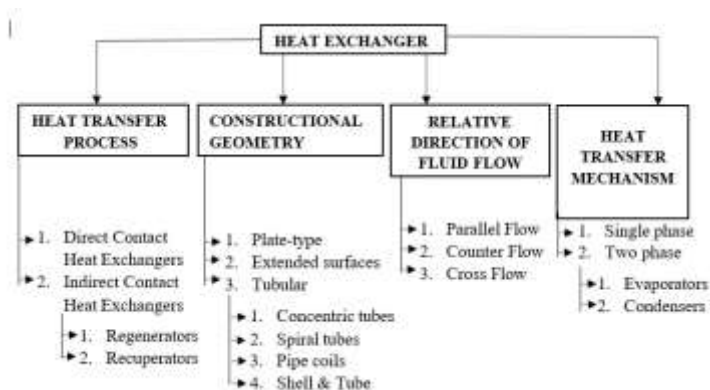


Figure 1: Classification of heat exchanger

The two fluid streams, one hot and one cold, move through the heat exchanger in the same direction in a parallel flow heat exchanger. One end of the heat exchanger receives the fluid streams, while the other end is where they exit [43-49]. It is evident from the heat exchanger's schematic and temperature profile that the fluid streams' temperature difference reduces as it moves from the entrance to the outlet of the parallel flow heat exchanger (Figure 2).

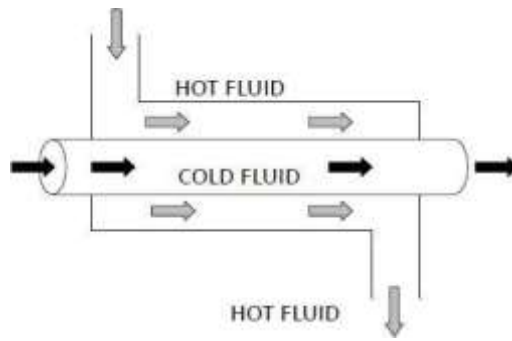


Figure 2: Parallel flow heat exchanger

The two fluid streams, one hot and one cold, move in nearly opposite directions in a heat exchanger known as a counterflow. The two streams of fluid meet at opposing points. The disparity in temperature between the two streams of fluid is very stable [50-55]. The maximum heat transfer rate for a specific surface area can be found using counterflow heat exchangers (Figure 3).

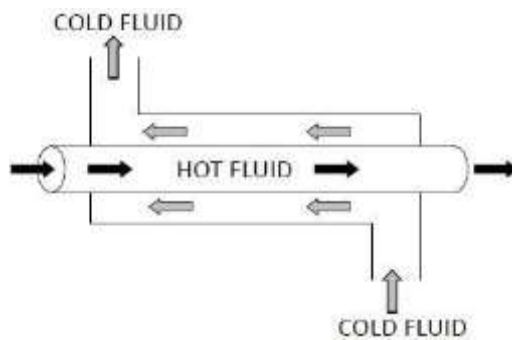


Figure 3: Counter flow heat exchanger

Typically, in a cross-flow heat exchanger, the two streams of fluid travel in opposite directions. The cross-flow heat exchanger is depicted schematically. Air is utilised as the cooling media in our laboratory's finned type cross-flow heat exchanger (Figure 4).

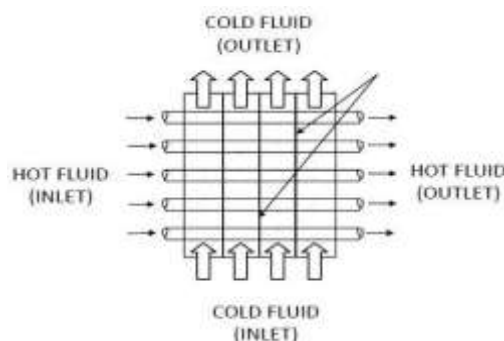


Figure 4: Cross-flow heat exchanger

The shell of this heat exchanger is typically cylindrical, and it encases a bundle of tubes. The orientation of the tubes is parallel to the axis of the shell. The tubes form a bundle, and one stream of fluid travels through them, while the other runs across the shell [56-60]. By utilising numerous shell & tube passes with baffles, the heat transmission between the fluid streams is improved. The fluid stream on the shell

side is redirected to flow in a reciprocating fashion over the tubes.

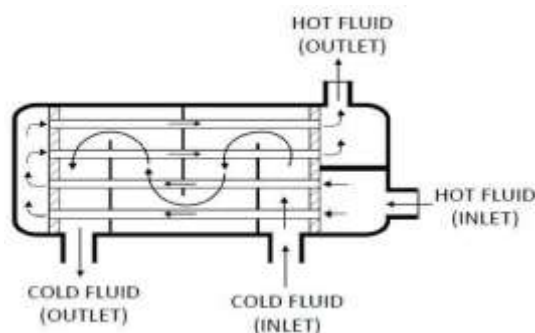


Figure 5: Shell & Tube heat exchanger

Plate heat exchangers are made up of a series of interconnected thin metal plates that have been corrugated (Figure 5). The fluid streams are distributed to the plates after entering the heat exchanger through the frame connections. Between each set of plates, there is a space that the two fluid streams flow through in opposite directions. Overall, the heat transfer between the fluid streams is improved because the metal plate grooves and tiny spacing create a turbulent flow [61-66]. Additionally, the flow's eddies clean the heat exchanger's surface, reducing fouling. Plate heat exchangers are widely used in industries due to their low cost, easy maintenance, and high thermal efficiency (Figure 6).

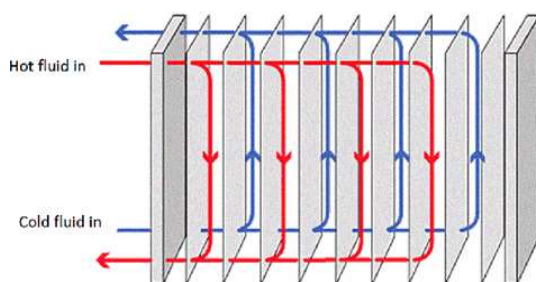


Figure 6: Plate-type heat exchange

To describe the application of infinitesimally "differential elements" in calculus and partial differential equations, the phrase "finite element" is used. Accurately determining values from analyses might be challenging, however this method helps with complicated structure analysis [67-81]. A more refined approach to conducting in-depth analyses of stresses, displacements, and support reactions in complicated shapes is made possible by increasing the number of small elements in MESH rather than the number of larger ones. This, in turn, produces more accurate conclusions. After 40 years, FEA has expanded its scope to include more complicated boundaries than finite differential equations [82]. When an agent specifies a software artefact to achieve goals with a collection of basic components that are subject to limitations, this is called software design. To design software is to engage in "all the work involved in envisioning, framing, implementing, commissioning, and eventually altering complex systems" or "the activity following requirements definition and preceding programming, as in a stylized software engineering process." Designing software typically entails figuring out problems and making plans for software solutions. Everything from low-level components to high-level architecture design and algorithm design falls under this category [83-89].

Literature Review

Radiators in cars use a circular heat exchanger, which Chavan and Tasgaonkar [1] describe. Because the air drawn in by the fan condenses into a small, circular region, the radiator's corners tend to get

very hot, necessitating a redesign of the heat exchanger to a more rectangular or square form. In the current layout, the power consumption of the fan is minimised because there is no heat transfer area in the centre, where the airflow is practically nonexistent. Create a round radiator with no corners for maximum efficiency.

The prismatic cells, also known as Li-ion cells, are addressed by Mayer et al. [2], who divide them into two cooled modules at the base. The top module incorporates a channel into its structure, while the bottom module makes use of cooling fins. Cylindrical cells are exposed to ambient air through the process of airflow via openings in the lateral cell surfaces. The fin allows heat to be transmitted from the battery pack to the air, and vice versa. Due to the fact that the extra weight of the fins exceeds the cooling benefits, using them to cool the electric car battery has become less popular.

The incorporation of fins was the primary emphasis of Kavitha et al. [3] in their investigation into methods to enhance and accelerate the cooling rate of radiators. Raising the contact surface area is the guiding idea of this approach. Altering the present fin geometry or creating new ones can enhance the contact surface, which in turn improves the heat dissipation rate. Radiators with broad contact surfaces, such as round fins, exhibit the highest temperature decrease compared to those with sharp or box fins, and the use of nanofluids as a coolant further enhances this heat dissipation. Improved convection between the nanoparticles and the surfaces of the base liquid gives it a high thermal conductivity.

The electronic controller for the cooling system was developed by Fatouh et al. [4]. The engine coolant temperature is controlled by the coolant flow rate and the speed of the radiator fan. The controller for the cooling system takes into account both linear quadratic regulation (LQR) and proportional integral derivative (PID) methods of control. The temperature of the engine's coolant can be adjusted using either the flow rate of coolant or the speed of the radiator fan. Depending on the engine load under various operating situations, to provide command control signals for the electric water pump and electric radiator fan.

In order to characterise the performance of the heat exchanger and to evaluate the present state of temperature control, Jignesh and Chaudhari [5] describe the cross-flow heat exchanger experiment setup as a helpful tool for analysing various parameters of finned tube heat exchangers and for testing heat loss. In general, a finned tube heat exchanger allows for faster heat transfer rates than one without one.

Problem Identification

- High thermal stress and decreased engine efficiency result from insufficient heat removal from the engine.
- It isn't the best cooling solution because it won't work in temperatures higher than 60 degrees Celsius.
- Keep the cooling fan and pump running as little as possible.
- Air contact surface decreases as contact surface area increases, resulting in subpar cooling.
- Make the radiator's block core larger.
- Tubes bent into a round profile.
- The astronomically priced circular radiators require the production of dies.

Research Gap

Many recent developments and investigations in engineering have focused on heat exchangers. A major breakthrough in engineering is also brought about by the analysis. As a result, we will be comparing heat exchangers manufactured of aluminium and copper using free and forced convection in our research. In order to find the heat transfer and total heat transfer coefficients, the log mean temperature difference (LMTD) approach was chosen. Compare the projected effectiveness from the NTU approach with the experimentally determined heat exchanger effectiveness [90-95]. However, no one has yet used finite element analysis to examine the materials. Now that we have done extensive research and study on the material, we can use finite element analysis to determine its mechanical and thermal properties (Figure 7).

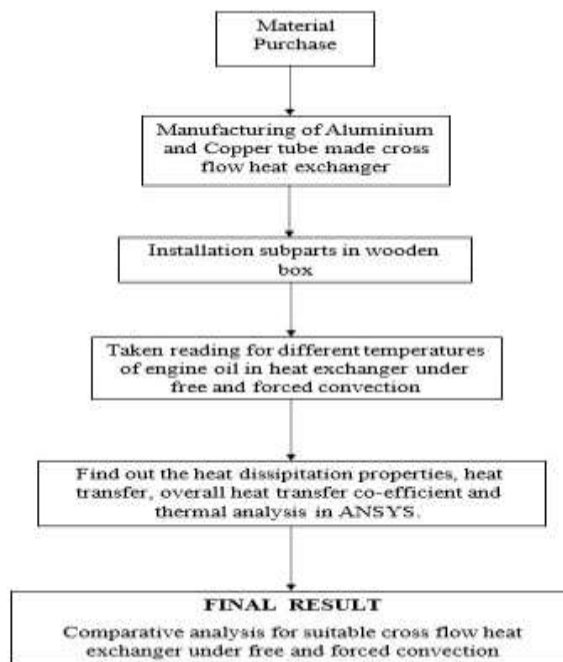


Figure 7: Fabrication process flow diagram

Mathematical Equations of Cross-Flow Heat Exchanger

There are two different method to determine the free and forced convection for aluminium and copper made heat exchanger.

• **FORCED CONVECTION:**

Heat transfer surface area, $A = 0.03749 \text{ m}^2$

Cross-section area of the air inlet, $A_c = 0.0319 \text{ m}^2$

Log mean temperature difference method:

the heat transfer between two fluids can be calculated from:

For Heat Transfer: $Q = F_r U A \Delta T_{lm}$

For heat transfer co-efficient:

Bulk mean temperature $T_m = \frac{T_{mi} + T_{mo}}{2}$

Reynolds number: $Re = \frac{UD}{\nu}$

Nusselt number: $Nu = \frac{hD}{k}$

Log mean temperature difference method:

$$(\Delta T)_m = \frac{(T_1 - t_2) - (T_2 - t_1)}{\ln \frac{(T_1 - t_2)}{(T_2 - t_1)}}$$

Xaxis value: $P = \frac{t_2 - t_1}{T_1 - t_1}$

Curve value R: $R = \frac{T_1 - T_2}{t_2 - t_1}$

Effectiveness-NTU Method

The effectiveness can be calculated form equation

Effectiveness of heat exchanger:

$$\epsilon = \frac{T_1 - T_2}{T_1 - t_1}$$

Capacity rate of Engine oil (Hot fluid):

$$C_h = \dot{m}_h \times C_{ph}$$

Capacity rate of Air (Cold fluid): $C_c = \dot{m}_c C_{pc}$

Curve $\frac{C_{min}}{C_{max}}$

Number of Transfer Units: $NTU = \frac{UA}{C_{min}}$

• **FREE CONVECTION:**

the heat transfer between two fluids can be calculated from: For

Heat Transfer: $Q = hA\Delta T = hA(T_w - T_\infty)$

Film Temperature: $T_f = \frac{T_w + T_\infty}{2}$

$$\beta = \frac{1}{T_{fin} \text{ K}}$$

Experimental Work

The thermal engineering laboratory of heat transfer in the mechanical engineering department of Dhaanish Ahmed College of Engineering designed and manufactured the cross-flow heat exchanger (Table 1).

Table 1: Reading of Copper radiator under Forced convection

No. of iteration	Engine oil inlet	Engine oil outlet	Air inlet	Air outlet	Mass flow rate	Engine oil flow velocity	Airflow
	in °C	in °C	in °C	in °C	in Kg/s	in m/s	in m/s
1	50	35.7	26.7	28.3	7.5×10^{-3}	0.137	5
2	71	56.9	28.14	30.4	0.0106	0.195	10
3	100	77.5	30.2	32.9	0.0136	0.256	15
4	125	109.7	31.4	33.8	0.01537	0.295	22

Heat transfer surface area, $A = 0.03749 \text{ m}^2$

The cross-section area of the air inlet, $A_c = 0.0319 \text{ m}^2$

To determine and verify the experimental heat transfer and engine oil temperature at the output, we have chosen to use a log mean temperature difference approach (Figure 8).

We know that, $T_1=50^\circ\text{C}$, $T_2=35.7^\circ\text{C}$, $t_1= 26.7^\circ\text{C}$, $t_2=28.3^\circ\text{C}$

Bulk mean temperature, $T_m = \frac{T_1+T_2}{2}$

$$T_m = \frac{50 + 35.7}{2} = 42.85^\circ\text{C}$$

$\rho=867.725\text{kg/m}^3$, $\gamma=7.77\times 10^{-3}\text{m}^2/\text{s}$, $k=0.1565\text{W/m}^2\text{k}$, $C_p=2.346\times 10^3 \text{ J/kgk}$

Reynolds number:
$$Re = \frac{UD}{\gamma} = \frac{.137 \times 9 \times 10^{-3}}{7.77 \times 10^{-5}} = 15.85$$

Since $Re < 2300$, Flow is laminar.

For laminar flow, Nusselt number, $Nu=3.66$

The Xaxis value is 0.06, and the curve value is 8.9375, the corresponding Y_{axis} value is 0.98, $F = 0.98$

Effectiveness-NTU Method

The effectiveness can be calculated form equation

Effectiveness of heat exchanger:

$$\epsilon = \frac{T_1-T_2}{T_1-t_1} = \frac{50-35.7}{50-26.7} = 0.536$$

(From HMT data book Page No. 161 (Ninth Edition))

Capacity rate of Engine oil (Hot fluid):

$$\begin{aligned} C_h &= \dot{m}_h \times C_{ph} \\ &= 7.548 \times 10^{-3} \times 2.546 \times 10^3 = 17.707 \text{ W/k} \end{aligned}$$

Capacity rate of Air (Cold fluid): $C_c = \dot{m}_c C_{pc}$

$$= 0.1768 \times 1.007 \times 10^3 = 178.0376 \text{ W/k}$$

$C_{min} = 17.707 \text{ W/k}$, $C_{max} = 178.0376 \text{ W/k}$

(To find NTU, refer HMT data book Page No. 174 (cross flow))

From graph, $Y_{axis} = \epsilon = 0.536$

$$\text{Curve} = \frac{C_{min}}{C_{max}} = \frac{17.707}{178.037} = 0.099$$

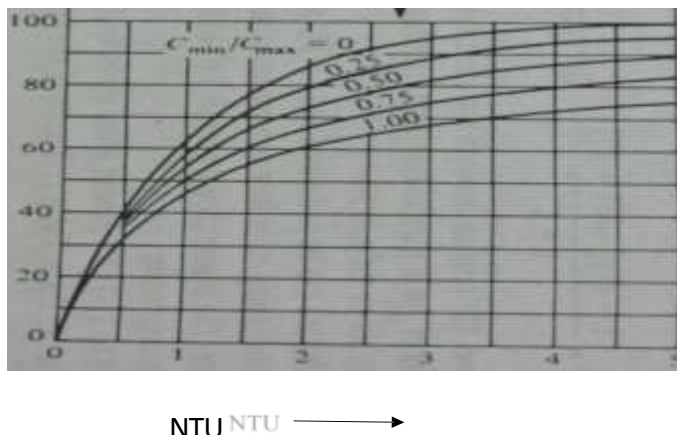


Figure 8: Effectiveness – Cross-flow, Both fluids unmixed for the copper radiator.

The corresponding Xaxis value is 0.89, NTU=0.89

Number of Transfer Units: NTU = UA

Cmin

$$0.89 = U \times 0.03749$$

$$17.707$$

$$U = 420.35 \text{ W/m}^2\text{K}$$

For Heat Transfer: $Q = FTUA\Delta T_{lm} = 0.98 \times 420.35 \times 0.03749 \times 14.4 = 222.389 \text{ W}$ (Table 2).

Table 2: Reading of Aluminum radiator under Forced convection

No. of iteration	Engine oil inlet	Engine oil outlet	Air inlet	Air outlet	Mass flow rate	Engine oil flow velocity	Airflow
	in °C	in °C	in °C	in °C	in Kg/s	in m/s	in m/s
1	50	42.9	27.1	29.5	5.83×10^{-3}	0.134	5
2	71	64.3	29.3	30.9	9.114×10^{-3}	0.213	10
3	100	81.4	28.9	29.7	0.0123	0.294	15
4	125	116.8	30.1	32.3	0.0148	0.359	22

Heat transfer surface area, $A = 0.04824 \text{ m}^2$

The cross-section area of the air inlet, $A_c = 0.0423 \text{ m}^2$

In order to determine and verify the experimental heat transfer and engine oil temperature at the outlet, the log mean temperature difference approach has been chosen:

Iteration1:

We know that, $T_1=50^\circ\text{C}$, $T_2=42.9^\circ\text{C}$, $t_1= 27.1^\circ\text{C}$, $t_2=29.5^\circ\text{C}$

Bulk mean temperature, $T_m = \frac{T_1+T_2}{2}$

$$T_m = \frac{50 + 42.9}{2} = 46.45^\circ\text{C}$$

Properties of Engine oil at 46.45°C :

(From HMT data book Page No. 33 (Ninth Edition))

$\rho=866.15\text{kg/m}^3$, $\gamma=6.97\times 10^{-5}\text{m}^2/\text{s}$, $k=0.155\text{W/m}^2\text{k}$, $C_p=2.297\times 10^3\text{J/Kgk}$

Reynolds number: $Re = \frac{UD}{\gamma} = .134\times 9\times 10^{-3}/6.97\times 10^{-5} = 15.36$

Since $Re<2300$, Flow is laminar.

For laminar flow, Nusselt number, $Nu=3.66$

(From HMT data book Page No. 132 (Ninth Edition))

Nusselt number: $Nu = \frac{hD}{k}$
 $3.66=h\times 9\times 10^{-3}/0.155$

Heat transfer Co-efficient, $h=70.91\text{W/m}^2\text{k}$.

This is cross flow, both fluids unmixed type heat exchanger

$$Q=F\tau UA\Delta T_{lm}$$

(From HMT data book Page No. 160 (Ninth Edition))

Log mean temperature difference method:

$$(\Delta T)_m = \frac{(T_1-t_2)-(T_2-t_1)}{\ln\frac{(T_1-t_2)}{(T_2-t_1)}} = \frac{(50-29.5)-(42.9-27.1)}{\ln\frac{(50-29.5)}{(42.9-27.1)}} = 18.04^\circ\text{C}$$

Page 170 of the HMT data book (Ninth Edition) (Single-pass cross flow heat exchanger - Both fluids unmixed) contains the correction factor F.

In order to resolve the analysis, we employ ANSYS 15. We begin by using the precise model dimensions to create the 3D geometry in Solidworks 13. Then, we proceed to add material data for the cross-flow heat exchanger made of aluminium and copper one by one. By utilising SOLIDWORKS, the precise proportions of the heat exchanger may be modelled by repurposing existing materials [96-101]. Before being imported into Ansys Fluent for thermal analysis, the modelled cross-flow tubes and fins are built in Solidworks (Figures 9 and 10).

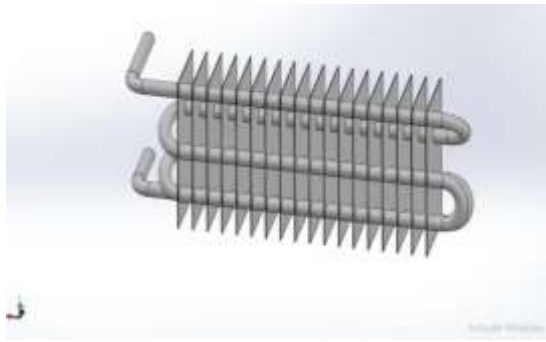


Figure 9: Copper tube radiator

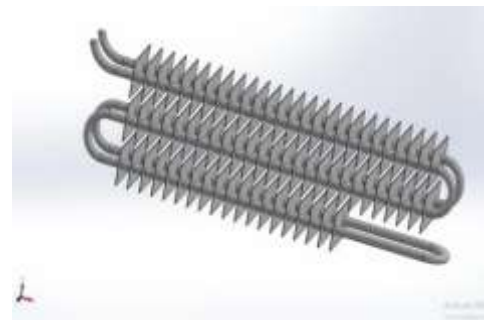


Figure 10: Aluminum tube radiator

The Finite Element Method, a numerical approach, is used in Finite Element Analysis (FEA), which simulates any physical process (FEM). An assembly's or part's behaviour under specified conditions can be predicted using finite element analysis (FEA). Modern simulation software is based on it, and engineers utilise it to uncover design weaknesses, tension points, and other issues. A colour scale that illustrates the distribution of pressure and temperature around the object, for example, is a common way to represent the results of FEA-based simulations. This study examines and presents the methods that was utilised to design the heat exchanger. As a result of

As part of the design process, the pressure drops, surface area for heat transfer, and cross-flow and tube-side heat transfer coefficients must be determined. In order to assess the part's rigidity under design pressures, the mechanical design of the heat exchanger incorporates calculations of the part's thickness, including the cross-flow channel, tube, and so on [102-107]. The thermal analysis behaviour of heat exchangers constructed of copper and aluminium is investigated in the ANSYS simulation by means of repetitive cooling system applications. Vehicle dynamics are required to ascertain the temperature variation with respect to vehicle speed. On the other hand, this research contrasted the results of free and forced convection analyses for radiator tubes constructed of aluminium and copper. After that, we use ANSYS to study the heat exchanger model that we created in Pro-Engineer. Here, oil is considered the hot fluid and air, the cold fluid, as none of these states undergoes phase change. As the coolant velocity increases, so does the heat transfer coefficient and the pressure drop. As the fluid's velocity increases, the thickness of the temperature boundary layer on the tube decreases [108-111]. A clear picture of the three-dimensional fluid flow can be obtained from flow trajectories, which display the flow streamlines.

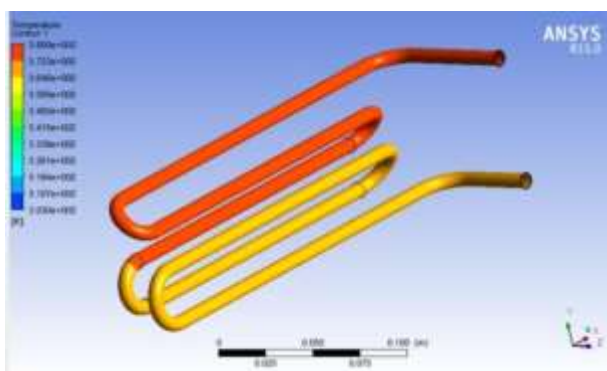


Figure 11: Heat transfer in copper

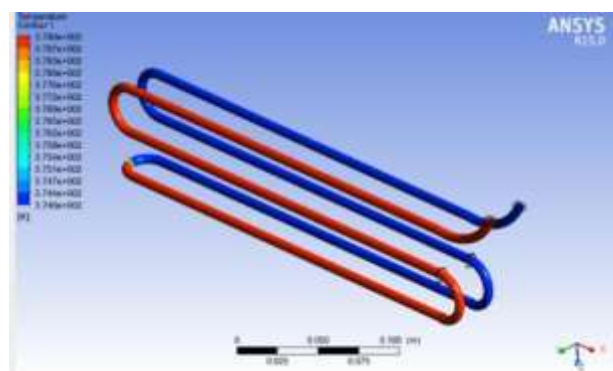


Figure 12: Heat transfer in aluminum Radiator

Radiator

The coolant flow temperatures at the intake and exit of the copper tube radiator are 380°K and 356.9°K, respectively, as shown in Figure 11. In the aluminium tube radiator, the corresponding temperatures are 379°K and 374°K, respectively, as shown in Figure 12.

Result and Discussion

The two fluids are kept separate as they flow at right angles to one another. As a result, heat is transferred from the coolant to the air. In cross-flow heat exchangers, this flow is used. As the coolant velocity increases, so does the heat transfer coefficient and the pressure drop. As the fluid's velocity increases, the thickness of the temperature boundary layer on the tube decreases. A clear picture of the three-dimensional fluid flow can be obtained from flow trajectories, which display the flow streamlines (Table 3).

Table 3: Results of Experimental Work in Copper and Aluminum Radiator

Description	Material of Flow Tube	Iteration	Results			
			Heat Transfer Coefficient	Overall Heat Transfer Coefficient	Heat Transfer Rate	Effectiveness
			in W/m ² K	in W/m ² K	in Watts	
Forced Convection	Copper	1	63.643	420.35	222.38	0.536
		2	60.59	301.436	380.65	0.328
		3	58.96	290.04	616.71	0.322
		4	57.34	280.22	907.09	0.16
	Aluminium	1	70.91	111.026	93.72	0.31
		2	67.02	104.75	185.84	0.26
		3	66.33	95.5	276.58	0.1616
		4	64.5	90.3217	386.36	0.086
Free Convection	Copper	1	63.44	406.812	218.06	0.545
		2	60.59	307.64	310.65	0.285
		3	58.96	294.1337	661.24	0.262
		4	57.34	239.178	789.02	0.124
		1	70.22	87.6	78.54	0.227
	Aluminum	2	67.893	104.97	193.02	0.215
3		45.04	140.67	425.29	0.203	
4		42.15	144.145	638.72	0.0518	

Moreover, that occurred because an increase in velocity causes a rise in the Reynolds number, which in turn increases the quantity of air that comes into contact with the surface of the heat exchanger in order to lower the temperature. When $Q=907.09\text{W}$ at $m=0.6288\text{kg/s}$, the maximum heat transfer occurs.

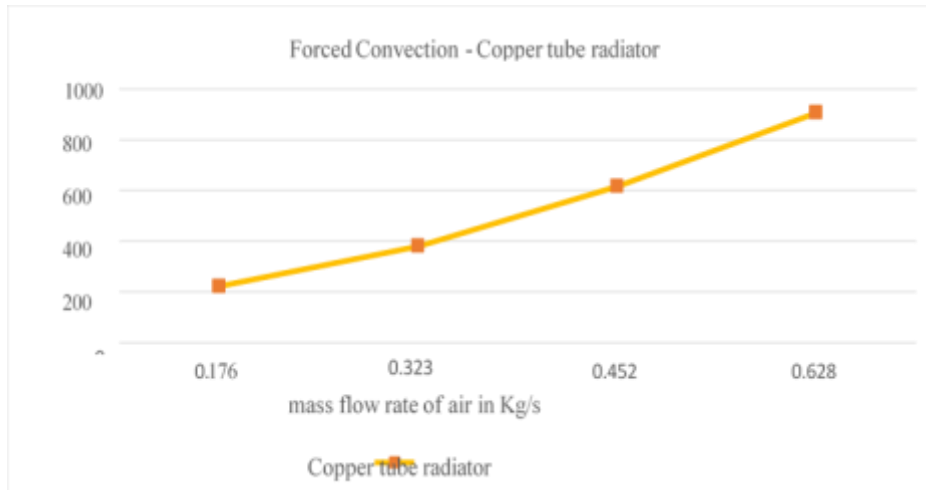


Figure 13: Relationship between heat transfer and mass flow rate of air with forced convection for Copper radiator

Figure 13 demonstrates that when the mass flow rate of air rose, heat transfer increased, when the engine oil flow rate is $5.83 \times 10^{-3} \text{Kg/s}$ at $Q=93.72 \text{ W}$. Moreover, that occurred because an increase in velocity causes a rise in the Reynolds number, which in turn increases the quantity of air that comes into contact with the surface of the heat exchanger in order to lower the temperature. $P=0.833 \text{ kg/s}$ yields a maximum heat transfer of $Q=386.36 \text{ W}$.

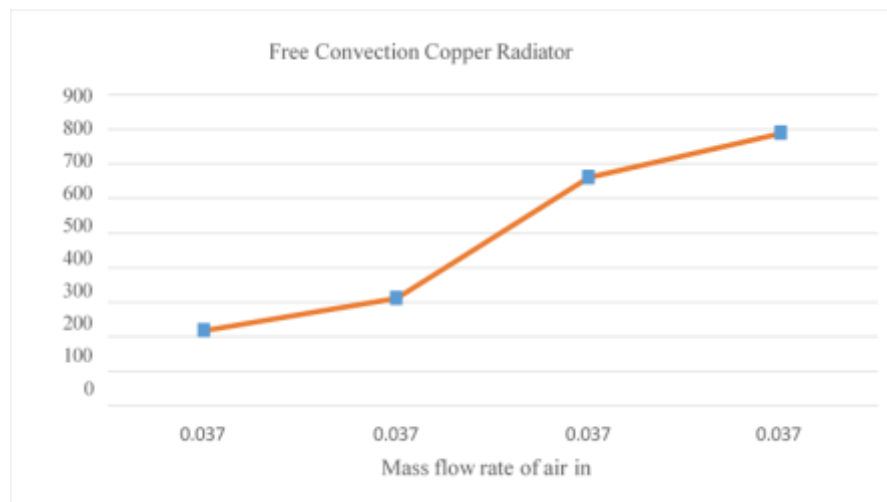


Figure 14: Relationship between heat transfer and mass flow rate of air with forced convection for Aluminum radiator

The heat transfer increased as the mass flow rate of air rose, as shown in Figure 14, when the engine oil flow rate is $7.4 \times 10^{-3} \text{Kg/s}$ and $Q=218.06 \text{ W}$. Moreover, that occurred because an increase in velocity causes a rise in the Reynolds number, which in turn increases the quantity of air that comes into contact with the surface of the heat exchanger in order to lower the temperature. At mass flow rate of 0.0377 kg/s , the maximum heat transfer is $Q=789.024 \text{ W}$.

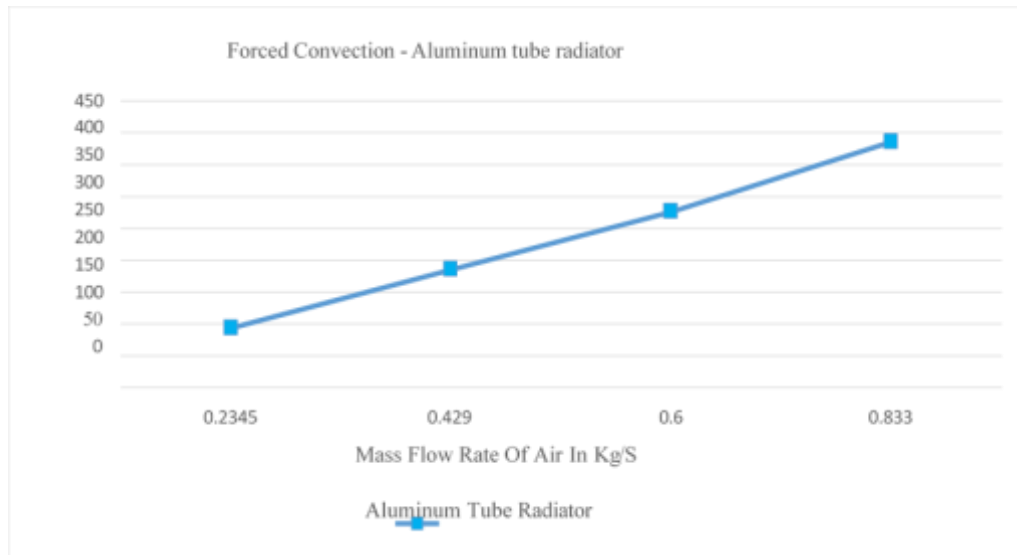


Figure 15: Relationship between heat transfer and mass flow rate of air with free convection for Copper radiator

A higher mass flow rate of air resulted in an increase in heat transfer, as seen in Figure 15, when the engine oil flow rate was $6.16 \times 10^{-3} \text{ Kg/s}$ and $Q=78.54 \text{ W}$. Moreover, that occurred because an increase in velocity causes a rise in the Reynolds number, which in turn increases the quantity of air that comes into contact with the surface of the heat exchanger in order to lower the temperature. When $m_a=0.05 \text{ kg/s}$, the maximum heat transfer is 638.72 W .

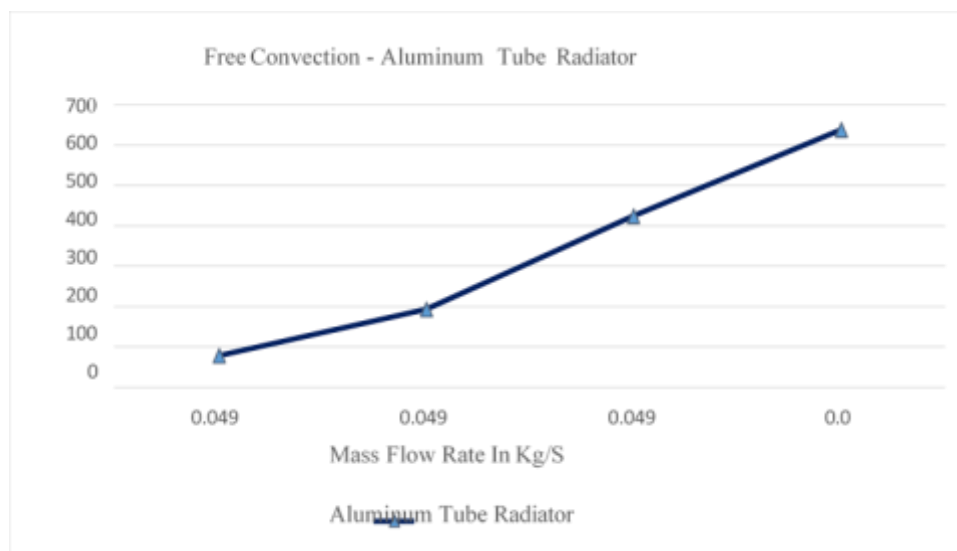


Figure 16: Relationship between heat transfer and mass flow rate of air with free convection for Aluminum radiator

As seen in Figure 16, the efficiency behaviour during the transfer of air mass is illustrated. The efficiency drops as the air mass flow rate rises, and Assuming an airflow rate (m_a) of 0.1768 kg/s , the minimum efficacy is determined to be 53.6% . Because, depending on the effectiveness connection, the

mass flow rate of air is inversely related to the effectiveness, that is what transpired. When the airflow rate (\dot{m}_a) was 0.6288 kg/s, the minimum effectiveness was 16 percent.

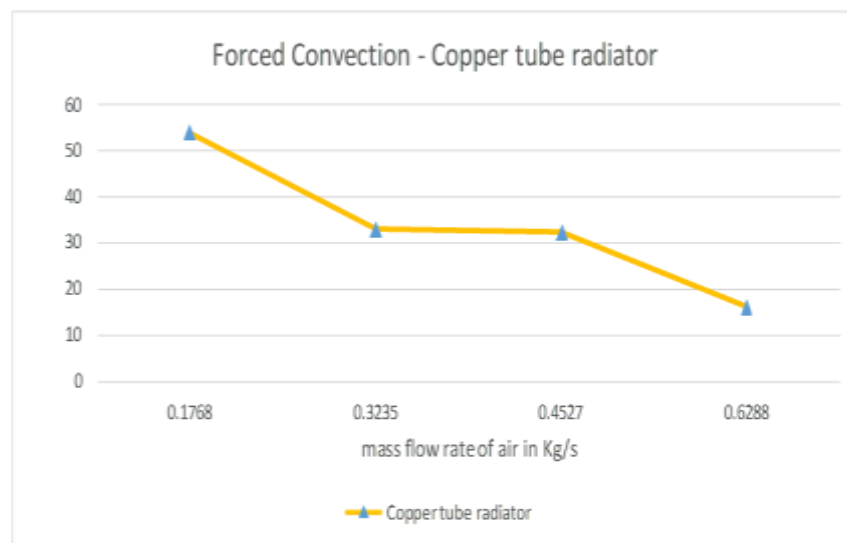


Figure 17: Relationship between effectiveness and mass flow rate with forced convection For Copper radiator

We used forced convection to build a copper tube cross-flow heat exchanger in a two-wheeler vehicle so that we could regulate the temperature of the engine oil and keep the protective film of lubricating oil from degrading and losing its lubricating capabilities (Figure 17). Heat exchangers can be found in many different types of machinery and appliances, including: boilers, condensers, evaporators, regenerators, oil coolers for heat engines, milk chillers in pasteurising facilities, radiators in automobiles, and many more industrial operations.

Conclusion

We find that the heat transfer rate is high when the copper tube radiator is subjected to forced convection, as shown by comparing the graphs obtained from the analysis with the free convection findings. The heat exchanger's efficiency in free convection with a copper tube radiator is improved by keeping an eye on the results of the thermal study. The structural analysis shows that heat exchangers operating under free convection have better heat dissipation characteristics than those operating under forced convection. The heat transfer and total heat transfer coefficients have been calculated using the log mean temperature difference approach, which is effective with respect to the fluid's inlet and outlet temperatures. Increasing the mass flow rate of air in structural analysis enhances heat transmission, as shown by this conclusion. More so, the efficacy diminishes as the air mass flow rate rises. In order to determine the rate of heat transfer, this study will compare and contrast current radiators. An Arduino screen, which shows the results of the tests on your computer, is a great addition to the lab. Consider utilising other heat exchangers. Gaining an understanding of the cooling engines you'll need, as well as the heat exchanger's efficiency and cooling capacity, is within your reach with this project.

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